Solve for convective film coefficient using: Principles of Heat Transfer by Kreith

mbar := 10^{-3} bar uPa := 10^{-6} Pa kJ := 1000J $T_i := (273.15 - 25) K \qquad T_i = 248.15 K \qquad \mu_V := 10.28 \mu Pa \cdot s \qquad \mu_{liq} := 267.5 \mu Pa \cdot s$ $c_{\text{liq}} \coloneqq 1019 \frac{J}{\text{kg} \cdot \text{K}}$ $\rho_{\text{liq}} \coloneqq 1565 \cdot \frac{\text{kg}}{\text{m}^3}$ $\rho_{\text{V}} \coloneqq 16.39 \frac{\text{kg}}{\text{m}^3}$ $k_v := 0.009 \frac{W}{m.K}$ $k_{liq} := 0.053 \frac{W}{m.K}$ $\sigma := .014 \frac{N}{m}$ surface tension at 247K $\mathbf{h}_{liq} \coloneqq 173.7 \frac{\mathrm{kJ}}{\mathrm{kg}} \qquad \mathbf{h}_{\mathrm{v}} \coloneqq 275.6 \frac{\mathrm{kJ}}{\mathrm{kg}} \qquad \Delta \mathbf{h} \coloneqq \mathbf{h}_{\mathrm{v}} - \mathbf{h}_{liq} \qquad \Delta \mathbf{h} = 101.9 \cdot \frac{\mathrm{kJ}}{\mathrm{kg}} \qquad \lambda \coloneqq \Delta \mathbf{h}$ Q := 240WL1 := 4mTriple U-Tube Length $x_0 := 0.85$ $mdot := \frac{Q}{(x_0 - x_i) \cdot \lambda}$ $mdot = 2.944 \times 10^{-3} \frac{kg}{s}$ $x_i := 0.05$ **Tube dimensions** t := .012in wall thickness $c_{h} := 4.9mm - 2 \cdot t$ $c_{r} := \frac{4.9}{2}mm - t$ $w_{c} := 2mm$ $A_{c} := w_{c} \cdot c_{h} + \pi \cdot c_{r}^{2}$ $P_c := 2 \cdot w_c + 2 \cdot \pi \cdot c_r$ $D_h := 4 \cdot \frac{A_c}{P}$ $D_h = 5.272 \cdot mm$ $A_t := A_c$ L1 = 4m $G1 := \frac{\text{mdot}}{A_t} \qquad G1 = 127.791 \frac{\text{kg}}{\text{m}^2 \text{ s}}$ $h_{\text{flux}} := \frac{240W}{P_{\text{o}} \cdot L1}$ $h_{\text{flux}} = 3.433 \times 10^3 \cdot \frac{W}{r_{\text{sat}}^2}$ $T_{\text{sat}} := T_{\text{i}}$ $T_{\text{sat}} = 248.15 \text{ K}$ fluid saturation temperature temperature $k_{W} := 200 \frac{W}{mK}$ aluminum tube wall for T_{sat} = 238K the saturation pressure is 1.67bar $P := 1.671 \text{bar} \qquad P = 1.671 \times 10^5 \text{Pa} \qquad Pr_{\text{liq}} := \frac{\mu_{\text{liq}} \cdot c_{\text{liq}}}{k_{\text{lic}}} \qquad Pr_{\text{liq}} = 5.143$ $\alpha_{liq} := \frac{k_{liq}}{\rho_{liq} \cdot c_{liq}} \qquad \alpha_{liq} = 3.323 \times 10^{-8} \frac{m^2}{s} \qquad \qquad \text{thermal diffusivity}$

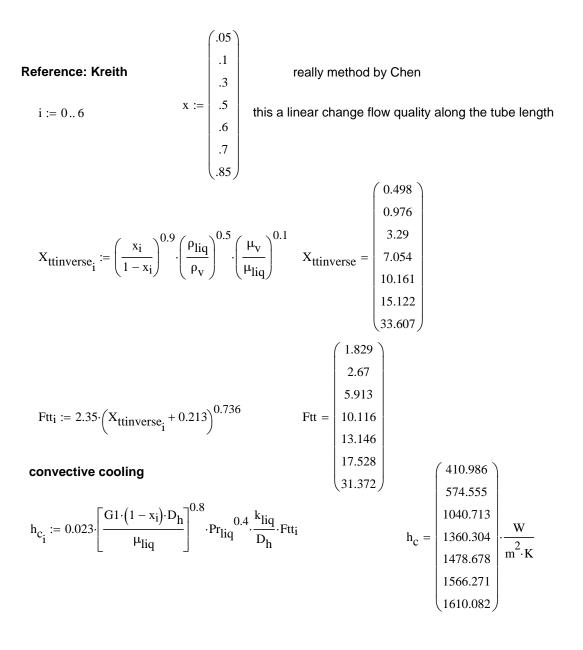
Condition for nucleate boiling to occur

$$\Delta T_{n} := \left(\frac{8 \cdot \sigma \cdot h_{flux} \cdot T_{sat}}{\lambda \cdot \rho_{v} \cdot k_{liq}}\right)^{0.5} \qquad \Delta T_{n} = 1.038 \,\mathrm{K}$$

any differential film temp above this

Two Phase Convection-4.9mm Oval Triple U-Tube C3F8_5_12_08.xmcd

W.O. Miller iTi

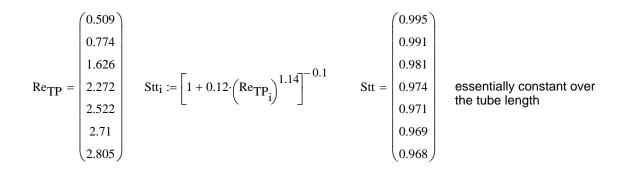


${\bf h}_{\rm c}$ is the cooling from forced convection. The next step is to calculate the cooling from evaporation.

Solving for evaporation requires assuming a temperature difference between the tube wall and the saturation temperature of the fluid. After calculating this contribution we can solve for this temperature difference through the combination of both cooling mechanisms. If the end result agrees with the assumption, the process stops otherwise the process is repeated with a new assumed ΔT_{sat}

$$\operatorname{Re}_{TP_{i}} \coloneqq \frac{G1 \cdot (1 - x_{i}) \cdot D_{h}}{\mu_{liq}} \cdot (Ftt_{i})^{1.25} \cdot 10^{-4} \qquad \text{needed to calculate } S_{tt}$$

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For the assumed ΔT_{sat} (wall temp-fluid saturation temp) we calculate the value of ΔP_{sat} for C3F8. This value is the change in saturation pressure corresponding to the change in fluid temperature. At -25C the change in saturation pressure per degree C is 6800Pa, or for 5C ΔT_{sat} , the change is 34000Pa

 $\Delta T_{sat} := 3.15 K$ temperature difference between wall and fluid, assumed to start the process.

$$\begin{split} \Delta p_{sat} &\coloneqq \Delta T_{sat} \cdot 6800 \frac{Pa}{K} \qquad \Delta p_{sat} = 214.2 \cdot \text{mbar} \qquad \begin{array}{l} \text{pressure difference for temperature} \\ h_{b_0} &\coloneqq .00122 \cdot \left(\frac{k_{liq} \stackrel{0.79}{\longrightarrow} \cdot c_{liq} \stackrel{0.45}{\longrightarrow} \cdot \rho_{liq} \stackrel{0.49}{\longrightarrow} \right) \cdot \Delta T_{sat} \stackrel{0.24}{\longrightarrow} \cdot \Delta p_{sat} \stackrel{0.75}{\longrightarrow} \cdot \text{Stt}_0 \qquad \begin{array}{l} \text{Chen} \\ \text{h}_{b_0} &= 679.893 \cdot \frac{W}{m^2 \cdot K} \qquad h_0 &\coloneqq h_{b_0} + h_{c_0} \qquad h_0 = 1.091 \times 10^3 \cdot \frac{W}{m^2 \cdot K} \end{split}$$

$$\Delta t_{b_0} := \frac{h_{\text{flux}}}{h_0} \qquad \Delta t_{b_0} = 3.147 \,\text{K} \qquad \Delta T \text{ assumed was } 3.15 \text{C}$$

For the next quality 0.1

<u>AT</u> = 2.87K temperature difference between wall and fluid, assumed to start the process.

$$\begin{split} & \Delta p_{sat} \coloneqq \Delta T_{sat} \cdot 6800 \frac{Pa}{K} \qquad \Delta p_{sat} = 195.16 \cdot mbar \qquad \begin{array}{l} & \text{pressure difference for temperature} \\ & h_{b_1} \coloneqq .00122 \cdot \left(\frac{k_{liq} \stackrel{0.79}{} \cdot c_{liq} \stackrel{0.45}{} \cdot \rho_{liq} \stackrel{0.49}{} \right) \\ & \sigma \stackrel{0.24}{} \cdot \rho_{w} \stackrel{0.24}{} \cdot \Delta p_{sat} \stackrel{0.75}{} \cdot Stt_1 \qquad \begin{array}{l} & \text{Chen} \\ & h_{b_1} = 618.072 \cdot \frac{W}{m^2 \cdot K} \qquad \qquad h_1 \coloneqq h_{b_1} + h_{c_1} \qquad h_1 = 1.193 \times 10^3 \cdot \frac{W}{m^2 \cdot K} \\ & \Delta t_{b_1} \coloneqq \frac{h_{flux}}{h_1} \qquad \Delta t_{b_1} = 2.878 \, K \qquad \Delta T \text{ assumed was } 2.87C \end{split}$$

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For the next quality 0.3

<u>AT_set</u> := 2.25K temperature difference between wall and fluid, assumed to start the process.

$$\Delta p_{sat} \coloneqq \Delta T_{sat} \cdot 6800 \frac{Pa}{K} \qquad \Delta p_{sat} = 153 \cdot \text{mbar} \qquad \begin{array}{l} \text{pressure difference for temperature} \\ \text{difference} \end{array}$$

$$h_{b_2} \coloneqq .00122 \cdot \left(\frac{k_{liq}^{0.79} \cdot c_{liq}^{0.45} \cdot \rho_{liq}^{0.49}}{\sigma^{0.5} \cdot \mu_{liq}^{0.29} \cdot \lambda^{0.24} \cdot \rho_{v}^{0.24}} \right) \cdot \Delta T_{sat}^{0.24} \cdot \Delta p_{sat}^{0.75} \cdot \text{Stt}_2 \qquad \text{Chen}$$

$$h_{b_2} = 480.713 \cdot \frac{W}{m^2 \cdot K}$$
 $h_2 := h_{b_2} + h_{c_2}$ $h_2 = 1.521 \times 10^3 \cdot \frac{W}{m^2 \cdot K}$

$$\Delta t_{b_2} \coloneqq \frac{h_{flux}}{h_2} \qquad \Delta t_{b_2} = 2.256\,K \qquad \Delta \text{T} \text{ assumed was } 2.5\text{C}$$

For the next quality 0.5

<u>MT_set.</u>:= 1.94K temperature difference between wall and fluid, assumed to start the process.

$$\begin{split} & \underbrace{\Delta p_{sat}}_{sat} \coloneqq \Delta T_{sat} \cdot 6800 \frac{Pa}{K} \qquad \Delta p_{sat} = 131.92 \cdot \text{mbar} \qquad \begin{array}{l} & \text{pressure difference for temperature} \\ & \text{h}_{b_3} \coloneqq .00122 \cdot \left(\frac{k_{liq}}{\sigma^{0.5} \cdot \mu_{liq}} \overset{0.79}{\circ} \cdot \overset{0.45}{\varsigma^{0.5} \cdot \mu_{liq}} \overset{0.49}{\circ} \cdot \overset{0.24}{\varsigma_{v}} \overset{0.24}{\circ} \cdot \overset{0.24}{\varsigma_{v}} \cdot \overset{0.24}{\varsigma_{v}} \cdot \overset{0.24}{\varsigma_{v}} \cdot \overset{0.75}{\varsigma_{v}} \cdot \text{Stt}_3 \qquad \begin{array}{l} \text{Chen} \\ & \text{h}_{b_3} = 411.904 \cdot \frac{W}{m^2 \cdot K} \qquad \qquad \begin{array}{l} h_3 \coloneqq h_3 \Rightarrow h_{c_3} \qquad h_3 = 1.772 \times 10^3 \cdot \frac{W}{m^2 \cdot K} \end{split}$$

 $\Delta t_{b_3} := \frac{h_{flux}}{h_3}$ $\Delta t_{b_3} = 1.937 \, \text{K}$ ΔT assumed was 1.94C

For the next quality 0.6

 $\Delta T_{\text{mat}} := 1.84 \text{K}$ temperature difference between wall and fluid, assumed to start the process.

$$\Delta p_{satv} \coloneqq \Delta T_{sat} \cdot 6800 \frac{Pa}{K} \qquad \Delta p_{sat} = 125.12 \cdot \text{mbar} \qquad \begin{array}{l} \text{pressure difference for temperature} \\ \text{difference} \end{array}$$

$$h_{b_4} \coloneqq .00122 \cdot \left(\frac{k_{liq}^{0.79} \cdot c_{liq}^{0.45} \cdot \rho_{liq}^{0.49}}{\sigma^{0.5} \cdot \mu_{liq}^{0.29} \cdot \lambda^{0.24} \cdot \rho_{v}^{0.24}} \right) \cdot \Delta T_{sat}^{0.24} \cdot \Delta p_{sat}^{0.75} \cdot \text{Stt}_{4}$$

$$h_{b_4} = 389.741 \cdot \frac{W}{m^2 \cdot K}$$
 $h_4 := h_{b_4} + h_{c_4}$ $h_4 = 1.868 \times 10^3 \cdot \frac{W}{m^2 \cdot K}$

$$\Delta t_{b_4} := \frac{h_{flux}}{h_4}$$
 $\Delta t_{b_4} = 1.837 \, \text{K}$ ΔT assumed was 1.84C

For the next quality 0.7

 $\Delta T_{\text{set}} := 1.77 \text{K}$ temperature difference between wall and fluid, assumed to start the process.

$$\Delta p_{sat} \coloneqq \Delta T_{sat} \cdot 6800 \frac{Pa}{K} \qquad \Delta p_{sat} = 120.36 \cdot \text{mbar} \qquad \begin{array}{l} \text{pressure difference for temperature} \\ \text{difference} \end{array}$$

$$h_{b_{5}} \coloneqq .00122 \cdot \left(\frac{k_{liq}^{0.79} \cdot c_{liq}^{0.45} \cdot \rho_{liq}^{0.49}}{\sigma^{0.5} \cdot \mu_{liq}^{0.29} \cdot \lambda^{0.24} \cdot \rho_{v}^{0.24}} \right) \cdot \Delta T_{sat}^{0.24} \cdot \Delta p_{sat}^{0.75} \cdot \text{Stt}_{5}$$

$$h_{b_5} = 374.248 \cdot \frac{W}{m^2 \cdot K}$$
 $h_5 := h_{b_5} + h_{c_5}$ $h_5 = 1.941 \times 10^3 \cdot \frac{W}{m^2 \cdot K}$

$$\Delta t_{b_5} := \frac{h_{flux}}{h_5} \qquad \Delta t_{b_5} = 1.769 \, \text{K} \qquad \Delta T \text{ assumed was } 1.77 \text{C}$$

For the next quality 0.85

 $\Delta T_{\text{sat}} := 1.74 \text{K}$ temperature difference between wall and fluid, assumed to start the process.

$$\Delta p_{sat} := \Delta T_{sat} \cdot 6800 \frac{Pa}{K} \qquad \Delta p_{sat} = 118.32 \cdot mbar \qquad \begin{array}{c} \text{pressure difference for temperature} \\ \text{difference} \end{array}$$

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W.O. Miller Convection Coefficient for Two Phase Flow with C3F8 at -25C LBNL-ATLAS iTi

$$\mathbf{h}_{b_{6}} \coloneqq .00122 \cdot \left(\frac{k_{liq}^{0.79} \cdot c_{liq}^{0.45} \cdot \rho_{liq}^{0.49}}{\sigma^{0.5} \cdot \mu_{liq}^{0.29} \cdot \lambda^{0.24} \cdot \rho_{v}^{0.24}} \right) \cdot \Delta T_{sat}^{0.24} \cdot \Delta p_{sat}^{0.75} \cdot Stt_{6}$$

$$h_{b_6} = 367.568 \cdot \frac{W}{m^2 \cdot K}$$
 $h_6 := h_{b_6} + h_{c_6}$ $h_6 = 1.978 \times 10^3 \cdot \frac{W}{m^2 \cdot K}$

$$\Delta t_{b_6} := \frac{h_{\text{flux}}}{h_6} \qquad \Delta t_{b_6} = 1.736 \,\text{K}$$

 ΔT assumed was 1.74C